



# International Journal for Innovative Engineering and Management Research

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Title: **HEAT TRANSFER CHARACTERISTICS OF PARABOLIC SOLAR TROUGH BY USING FEA**

Volume 06, Issue 11, Pages: 364–370.

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## HEAT TRANSFER CHARACTERISTICS OF PARABOLIC SOLAR TROUGH BY USING FEA

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**Abstract**— Solar energy currently represents the most abundant inexhaustible, nonpolluting, and free energy resources that could have a positive meaning in alleviating the global energy shortage and environmental pollution. However, the solar energy is intermittent in nature and that received on earth is of small flux density due to atmospheric scattering and abortion, making it necessary to use large surfaces to collect solar energy for drying cut tobacco process utilization. The major technologies used for solar energy conversion to heat are thermal processes comprising of solar collectors. In this thesis the fluid flow through solar collectors( flat plate and parabolic trough) is modeled using CREO design software. The thesis will focus on thermal and CFD analysis with different fluids air, water,R30 and R60 of the solar collectors. Thermal analysis done for the solar collectors by steel, aluminum & copper materials.In this paper, CFD analysis to determine the heat transfer coefficient, heat transfer rate, mass flow rate, pressure drop at different fluids and thermal analysis to determine the temperature distribution, heat flux with different materials.3D modeled in parametric software Pro-Engineer and analysis done in ANSYS

**Keywords:** Solar collector, FEA, CFD, ANSYS.

### I. INTRODUCTION

The operation of any solar thermal energy collector can be described as an energy balance between the solar energy absorbed by the collector and the thermal energy removed or lost from the collector. If no alternative mechanism is provided for removal of thermal energy, the collector receiver heat loss must equal the absorbed solar Energy. The temperature of the receiver increases until the convective and radiation heat loss from the receiver Equals the absorbed solar energy. The temperature at which

this occurs is termed the collector stagnation Temperature. For control of the collector temperature at some point cooler than the stagnation temperature, active removal of heat must be employed. This heat will then available for use in a solar energy system. The rate at which Heat is actively removed from the collector determines the collector operating temperature. For removal of a large Fraction of the absorbed solar energy as useful heat, the amount of heat

lost from the receiver must be kept small. Receiver heat loss can be reduced by operating the collector near the ambient temperature (such as with low-temperature flat-plate collectors) or by constructing the collector such that heat loss at elevated temperature is reduced. The most common way of reducing receiver heat loss at elevated temperatures is to reduce the size of the hot surface (i.e., the receiver) since heat loss is directly proportional to area of the hot surface. Concentrating collectors reduce the area of the receiver by reflecting (or refracting) the light incident on a large area (the collector aperture) over an absorber of small area. With reduced heat loss, concentrating collectors can operate at elevated temperatures and still provide significant quantities of useful thermal energy.

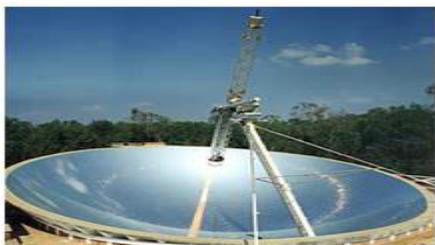


Fig. Solar Thermal Bowl Collector

## II. ANALYSIS OF PARABOLIC SOLAR TROUGH

A novel parabolic trough concentrating solar heating for cut tobacco drying system was established. The opening width effect of V type metal cavity absorber was investigated. A cut tobacco drying mathematical model calculated by fourth-order Runge-Kutta numerical solution method was used to simulate the cut tobacco drying process. And finally the orthogonal test method was used to optimize the parameters of cut tobacco drying process. The result shows that the heating rate, acquisition factor, and collector system efficiency increase with increasing the opening width of the absorber. The simulation

results are in good agreement with experimental data for cut tobacco drying process. The relative errors between simulated and experimental values are less than 8% ,indicating that this mathematical model is accurate for the cut tobacco airflow drying process. The optimum preparation conditions are an inlet airflow velocity of 15 m/s, an initial cut tobacco moisture content of 26%, and an inlet airflow temperature of 200°C. The thermal efficiency of the dryer and the final cut tobacco moisture content are 66.32% and 14.15%, respectively. The result shows that this parabolic trough concentrating solar heating will be one of the heat recourse candidates for cut tobacco drying system.

### INTRODUCTION TO CAD

Computers are being used increasingly for both design and detailing of engineering components in the drawing office. Computer-aided design (CAD) is defined as the application of computers and graphics software to aid or enhance the product design from conceptualization to documentation. CAD is most commonly associated with the use of an interactive computer graphics system, referred to as a CAD system.

### INTRODUCTION TO PRO/ENGINEER

**Pro/ENGINEER, PTC's parametric, integrated 3D CAD/CAM/CAE solution**, is used by discrete manufacturers for mechanical engineering, design and manufacturing. This powerful and rich design approach is used by companies whose product strategy is family-based or platform-driven, where a prescriptive design strategy is critical to the success of the design process by embedding engineering constraints and relationships to quickly optimize the design, or where the resulting geometry may be complex or based upon equations. Pro/ENGINEER provides a complete

set of design, analysis and manufacturing capabilities on one, integral, scalable platform.

## INTRODUCTION TO FEA

FEA consists of a computer model of a material or design that is stressed and analyzed for specific results. It is used in new product design, and existing product refinement. A company is able to verify a proposed design will be able to perform to the client's specifications prior to manufacturing or construction. Modifying an existing product or structure is utilized to qualify the product or structure for a new service condition. In case of structural failure, FEA may be used to help determine the design modifications to meet the new condition.

## INTRODUCTION TO ANSYS

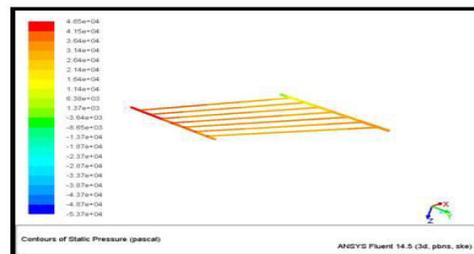
ANSYS is general-purpose finite element analysis (FEA) software package. Finite Element Analysis is a numerical method of deconstructing a complex system into very small pieces (of user-designated size) called elements. The software implements equations that govern the behaviour of these elements and solves them all; creating a comprehensive explanation of how the system acts as a whole. These results then can be presented in tabulated, or graphical forms. This type of analysis is typically used for the design and optimization of a system far too complex to analyze by hand. Systems that may fit into this category are too complex due to their geometry, scale, or governing equations. ANSYS provides a cost-effective way to explore the performance of products or processes in a virtual environment. This type of product development is termed virtual prototyping.

## INTRODUCTION TO CFD

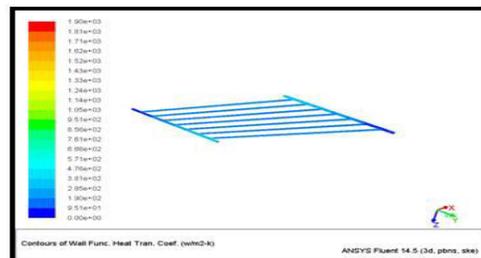
Computational fluid dynamics, usually abbreviated as CFD, is a branch of fluid mechanics that uses numerical methods and

algorithms to solve and analyze problems that involve fluid flows. Computers are used to perform the calculations required to simulate the interaction of liquids and gases with surfaces defined by boundary conditions. With high-speed supercomputers, better solutions can be achieved. Ongoing research yields software that improves the accuracy and speed of complex simulation scenarios such as transonic or turbulent flows. Initial experimental validation of such software is performed using a wind tunnel with the final validation coming in full-scale testing, e.g. flight tests.

## III. EXPERIMENTAL RESULTS



## HEAT TRANSFER COEFFICIENT

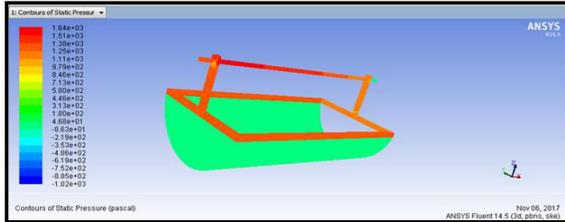


## MASS FLOW RATE & HEAT TRANSFER

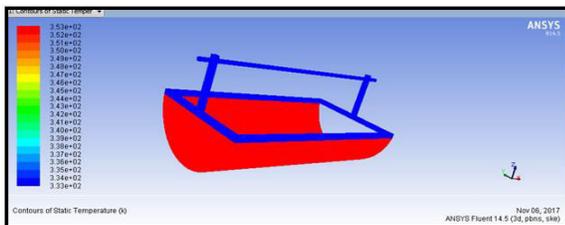
Mass Flow Rate		(kg/s)
inlet	0.010499999	
interior-partbody	0.026461512	
outlet	-0.010539424	
wall-partbody	0	
Net	-3.9424747e-05	
Total Heat Transfer Rate		(w)
inlet	790.97839	
outlet	-793.94824	
wall-partbody	0	
Net	-2.9698486	

## CFD ANALYSIS OF PARABOLIC SOLAR TROUGH

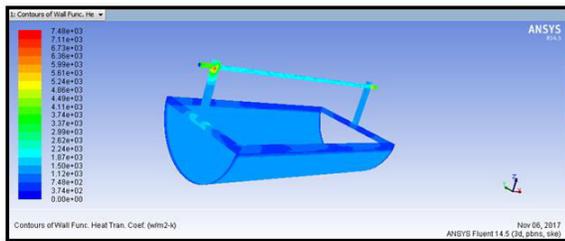
## CASE 1-ORIGINAL MODEL FLUID -WATER STATIC PRESSURE



## TEMPERATURE



## HEAT TRANSFER CO-EFFICIENT



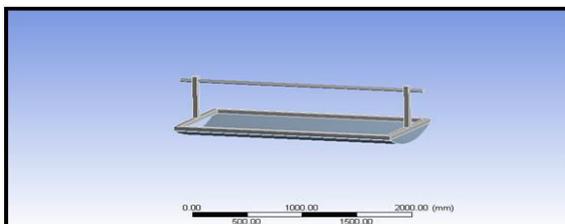
## MASS FLOW RATE

Mass Flow Rate	(kg/s)
inlet	1.4999995
interior- msbr	-9.4357023
outlet	-1.5012258
wall- msbr	0
Net	-0.001226306

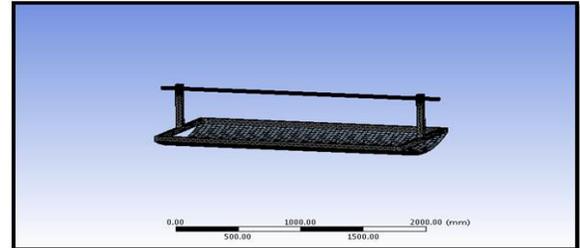
## HEAT TRANSFER RATE

Total Heat Transfer Rate	(w)
inlet	218614.02
outlet	-219101.16
wall- msbr	310.48871
Net	-176.65192

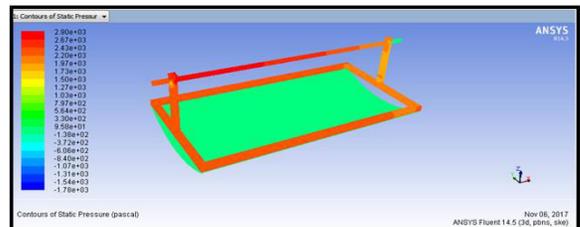
## CASE 2-MODIFIED MODEL IMPORTED MODEL



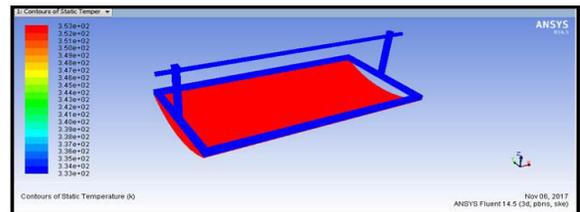
## MESHED MODEL



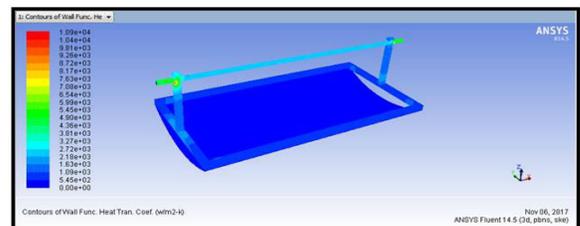
## FLUID -WATER STATIC PRESSURE



## TEMPERATURE



## HEAT TRANSFER CO-EFFICIENT



## MASS FLOW RATE

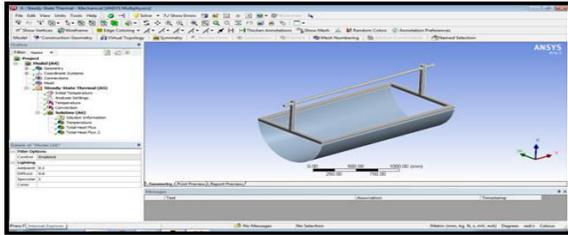
Mass Flow Rate	(kg/s)
inlet	1.9999996
interior- msbr	42.424152
outlet	-2.0063186
wall- msbr	0
Net	-0.0063189268

## HEAT TRANSFER RATE

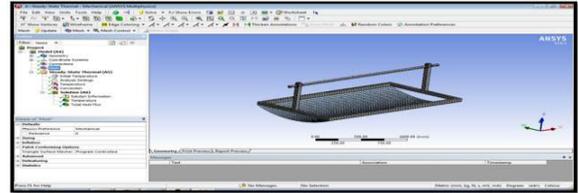
Total Heat Transfer Rate	(w)
inlet	291485.38
outlet	-292720.47
wall- msbr	311.07779
Net	-924.01596

## THERMAL ANALYSIS OF SOLAR PARABOLIC TROUGH

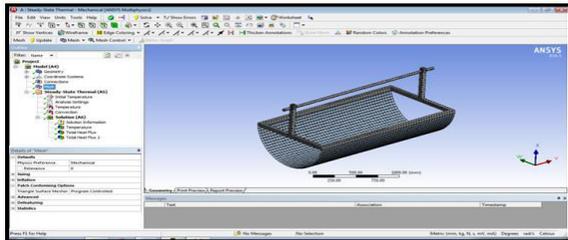
**IMPORTED MODEL**



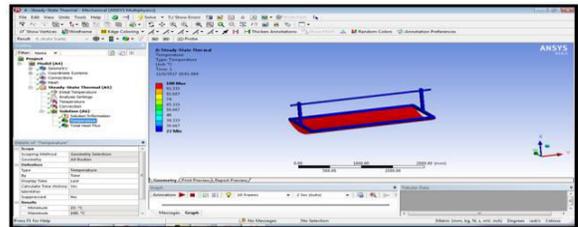
**MESHED MODEL**



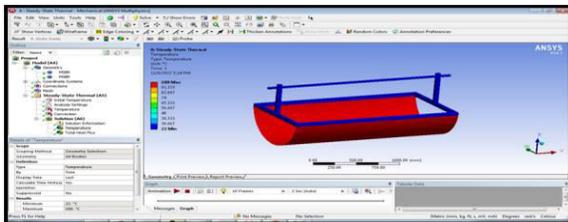
**MESHED MODEL**



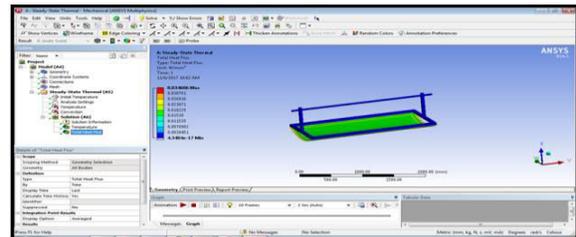
**MATERIAL- STEEL TEMPERATURE**



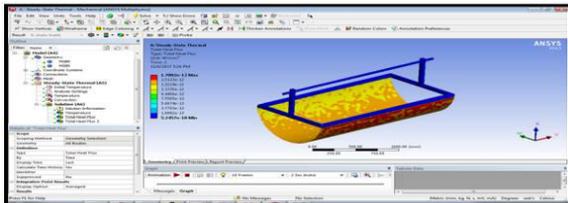
**BOUNDARY CONDITIONS CASE -1 ORIGINAL MODEL MATERIAL- STEEL TEMPERATURE**



**HEATFLUX**



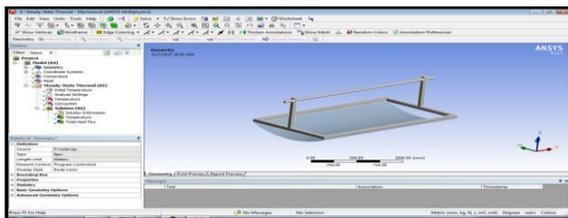
**HEAT FLUX**



**RESULTS TABELS CFD ANALYSIS OF SOLAR FLAT PLATE**

Fluids	Pressure (Pa)	Heat transfer coefficient (w/m <sup>2</sup> -k)	Mass flow rate (kg/s)	Heat transfer rate(w)
AIR	3.55e+04	1.44e+03	1.0958e-05	0.82574
WATER	4.65e+04	1.90e+03	3.9424e-05	2.9698
R30	4.68e+04	1.64e+03	2.5503e-05	1.920105
R160	3.70e+04	1.22e+03	5.0231e-05	3.1235e-05

**CASE -2 MODIFIED MODEL IMPORTED MODEL**



**CFD ANALYSIS RESULTS ORIGINAL MODEL**

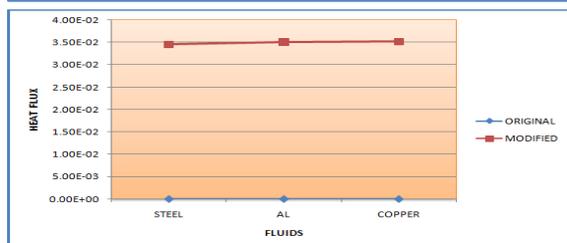
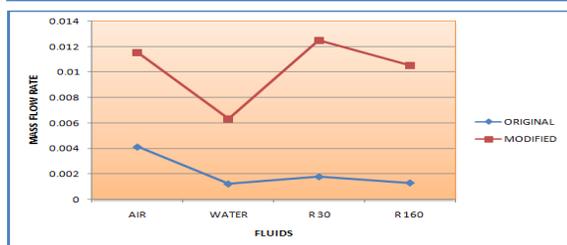
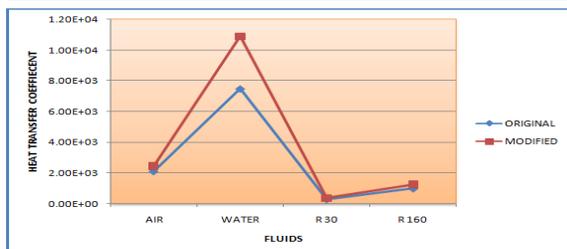
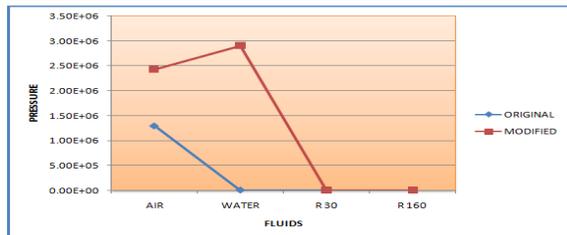
FLUID	PRESSURE (pa)	HEAT TRANSFER COEFFICIENT	MASS FLOW RATE	HEAT TRANSFER RATE
AIR	1.30E+06	2.11E+03	0.00412	150.8656
WATER	1.64E+03	7.48E+03	0.0012263	176.651
R 30	1.24E+03	2.87E+02	0.001788	48.89
R 160	1.79E+03	9.84E+02	0.0013078	51.32959

## MODIFIED MODEL

FLUID	PRESSURE (pa)	HEAT TRANSFER COEFFICIENT	MASS FLOW RATE	HEAT TRANSFER RATE
AIR	2.43E+06	2.45E+03	0.011516	403.288
WATER	2.90E+06	1.09E+04	0.0063189	924.015
R 30	2.07E+03	3.60E+02	0.01248	449.7378
R 160	3.21E+03	1.25E+03	0.010541	371.2136

## THERMAL ANALYSIS RESULTS

CASES	MATERIAL	HEAT FLUX
ORIGINAL MODEL	STEEL	1.7002E-12
	ALUMINUM ALLOY	4.5574E-12
	COPPER	1.237E-11
MODIFIED MODEL	STEEL	0.034606
	ALUMINUM ALLOY	0.035126
	COPPER	0.035206



## IV. CONCLUSIONS

In this paper, fluid flow through solar collectors (flat plate and parabolic trough) is modeled using CREO design software. The thesis will focus on thermal and CFD analysis with different fluids air, water, R30 and R60 of the solar collectors. Thermal analysis done for the solar collectors by steel, aluminum & copper materials. By observing the CFD analysis the pressure drop & velocity values are more for water fluid at solar parabolic trough collectors compared with flat plate collector. The more heat transfer rate at fluid water. By observing the thermal analysis Heat flux value is less for steel material than aluminum and copper at solar collectors, and heat flux value is more for copper material than alluminium and steel at solar absorber. So we can conclude that steel material is better for solar collectors and copper material is better for absorber and water is the best working fluid.

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